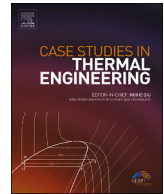




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Desirability-based optimization of dual-fuel diesel engine using acetylene as an alternative fuel

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HIGHLIGHTS

- Engine performance and emissions fuelled with acetylene/diesel were analyzed.
- The box-Behnken design was employed for the Design of Experiments.
- 4.48 lpm AGFR, 27.1 % BTE, and 76.58 bar Pmax are the optimized results.
- Tweedie has a better R² and MSE performance than LightGBM.

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ABSTRACT

The study examined the dual-fuel engine performance employing acetylene gas as primary fuel and diesel as pilot fuel. The engine's operational parameters were adjusted using the Box-Behnken design, and the results were recorded. The best operating settings were yielded as 81.25 % engine load, 4.48 lpm acetylene gas flow rate and the compression ratio were 18. At this optimized setting the BTE was 27.1 % and the engine emitted 360 ppm of NO_x, 56.2 ppm of HC, 104 ppm of CO. The experimental data at optimized setting was compared to the optimized results, and the percentage of errors was within 7 %. Two advanced machine learning methods, LightGBM and Tweedie, were used to predict engine efficiency and emissions. Tweedie-based models had an R² value of 0.89–1, while LightGBM-based models had an R² value of 0.38–1. The mean squared error was 0.24–45.04 for Tweedie-based models and 8.5 to 153.89 for LightGBM-based models. On the basis of R² and MSE, it was observed that Tweedie was superior at making predictions than LightGBM. The study demonstrated the efficient functioning of a dual-fuel engine using acetylene as an alternative fuel for increased performance and lower emissions.

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List of nomenclature

AGFR	Acetylene Gas Flow Rate
ANN	Artificial Neural Networks
BDC	Bottom Dead Centre
BTE	Brake Thermal Efficiency
CCD	Central Composite Design
CO	Carbon Monoxide
CO ₂	Carbon Dioxide
CR	Compression Ratio
C ₂ H ₂	Acetylene
DoE	Design of Experiment
DF	Dual-fuel
ER	Equivalence ratio
GFR	Gas flow rate
GHG	Greenhouse Gases
ICEs	Internal Combustion Engines
H ₂	Hydrogen
HC	Hydrocarbon
LCV	Lower Calorific Value
LightGBM	Light Gradient Boosting Machine
LPM	Liters per Minute
m	Mass
MSE	Mean Squared Error
ML	Machine Learning
MAPE	Mean Absolute Percentage Error
MAE	Mean Absolute Error
MBT	Minimum Spark Timing for Best Torque
NO _x	Oxides of Nitrogen
P	Pressure
P _{max}	Peak Cylinder Pressure
R ²	Coefficient of determinations
RE	Relative Error
RSM	Response Surface Methodology
SI	Spark Ignition
T	Temperature
TDC	Top Dead Centre
LHV	Lower heating value
UHC	Unburnt Hydrocarbons
USD	United States Dollars

1. Introduction

Nowadays, humans are abusing natural resources at an alarmingly high pace to make their lives more pleasant and comfortable [1,2], in which humans are constructing heavy industries to obtain a high level of comfort, which has resulted in a multiplication of energy requirements in recent years [3,4]. In addition, rapid urbanization and industrialization along with the fast-growing human population are resulting in shortage of energy sources and environmental pollution [5–7]. Due to this reason, massive deforestation and thorough exploitation of natural sources are necessary to meet enormous energy demand [8,9], which raises serious concerns about global warming, declining natural assets that are limited in nature, energy security, and environmental concerns [10,11]. In addition, massive deforestation along with the use of fossil fuels for energy and power production are found to increase CO₂ and greenhouse gas emissions [12,13], resulting in a serious climate change in the world. Facing the depletion of fossil fuels and climate change, society has begun to consider eco-friendly development which needs high technological advancements relating to alternative fuel and renewable energy sources to slow down this consumption of fossils especially in the transportation sector as there is leap and bound progress in the number of vehicles [14–16]. As a result, finding alternate fuels and renewable energy is critical to compensate the ever-increasing shortage of energy and reduce the climate change, as well as target sustainable development goals of nations and areas [17–19].

Indeed, with the continuous depletion of fossil fuels and environmental damage, experts all over the world are looking for viable replacement fuels [20,21] aiming to satisfy two main goals of diversification of provided fuels and reduction of pollutant emissions [22,23]. As reported in the literature, there have been many type of potential alternative fuels such as biodiesel [24,25], biogas/syn-

gas [26,27], bio-oil from pyrolysis [28], alcohol (methanol, ethanol, propanol, butanol) [29–31], furan [32,33], ether [34,35], hydrogen [36,37], natural gas [38,39], and others. As it can be created using non-fossil sources like calcium carbide and water, acetylene (C_2H_2) has been proposed as a potential replacement for fossil source-based fuels [40,41]. Acetylene generation depends on the process used, the price of the raw materials, the price of the energy, and other aspects. As of 2023, it's estimated using data from the local market that using calcium carbide to react with water will generate acetylene at a cheap price of about 0.4–0.5 USD per cubic meter. Contrary to what has been observed, acetylene is highly explosive, so flashback arrestor use is always advisable as a safety measure. Acetylene has properties such as being a colorless gas with a distinct garlic-like odour, being highly flammable, having an ignition temperature around 305 °C, having a high flame speed, etc. Due to all these distinct physical and chemical characteristics, acetylene is an intriguing prospective fuel for internal combustion engines (ICEs) [42,43].

Acetylene is one such fuel that can be used in ICEs because acetylene has a rapid speed of flame as well as low ignition energy, which can result in enhanced combustion efficiency and lower pollution emissions [44,45]. Acetylene has the potential to be a fuel for high-performance engines, including motorcycle, and racing vehicle engines, due to its high calorific value and fast flame [46,47]. Acetylene as a fuel for transportation is still in the trial stage, and no nation has widely adopted it because of its low energy density [48]. It is also quite challenging to store and transport acetylene because this gas is prone to breaking down at higher temperatures and pressures, and hence it is considered a highly unstable gas, which reduces its economic viability as a fuel for transportation [49]. Acetone-soaked porous materials like diatomaceous earth or activated charcoal are used to fill the cylinders used to store and transport acetylene. The acetone dissolves the acetylene and prevents it from breaking down, which aids in stabilizing it [50].

Acetylene as an alternative fuel for ICEs has been the subject of several experimental research in the past. ilhak et al. [51] experimentally carried out a performance and exhaust emission analysis of four-stroke water-cooled SI engine fuel using a mixture of gasoline and acetylene at various loads and flow rates of acetylene. It has been noted that with the induction of acetylene, not only does the overall thermal efficiency reduce, but there is also a significant drop in hydrocarbon emissions at all loads and a modest rise in NO at low loads. To reduce NO_x, Hilden et al. [52] studied the engine performance and exhaust emission behavior of an SI engine with acetylene as the fuel. This study team maintained consistent engine speed, constant airflow, and MBT spark timing throughout the studies. Equivalence ratios between 0.43 and 0.53 and compression ratios between 4 and 6 are ideal because preignition began at compression ratios higher than 6. Although acetylene fuelling results in a decrease in power and thermal efficiency when compared to petrol fuel, it also generates little pollutants. In addition, ilhak et al. [53] advocated that ethanol and acetylene would be good substitutes for petrol as they can be easily derived from renewable sources and thus evaluated this fuel along with petrol for exhaust emission and performance of a multi-cylinder, four-stroke, water-cooled Ford SI engine at different load settings at a constant speed. They found that NO_x formation and UHC emissions were reduced to a very large extent with this ethanol and acetylene. Because this acetylene has a low octane number of 50, it is susceptible to knocking, which can be overcome by supplying a lean mixture. Choudhary et al. [54] tested the engine for the performance of a diesel-acetylene fuel combination on a single-cylinder water-cooled four-stroke CI engine with a variable compression ratio. It has been reported that engines operate better at greater compression ratios of 19.5:1, although emission characteristics are better at lower compression ratios of 18:1. Sharma et al. [55] conducted an extensive study in India to identify the optimal flow rate of acetylene between 50 and 200 LPM that best suited the engine characteristics of SI engines driven by a combination of petrol and acetylene. The results mimic that at a flow rate of 100 LPM acetylene provides very little emission and hence using this optimum fuel flow rate various engine performances and exhaust emission characterization have been carried out. It has also been discovered that small anomalous acetylene combustion occurs exclusively at greater loads. The research findings of this research group stated that acetylene has a strong ability to replace petrol in the near future. Gupta et al. [42] turned a four-stroke SI engine into a six-stroke engine by injecting water at the end of the recompression stroke to assess and compare the performance of petrol and acetylene as fuels independently. When utilized in a six-stroke engine, acetylene exhibited encouraging results when compared to petrol, with a 45 % increase in thermal efficiency and a considerable reduction in exhaust emissions when compared to a four-stroke engine. The results of the exhaustive experimental examination showed that combining acetylene with the modified six-stroke engine increased engine performance and emissions. In light of its combustion properties, Lakshmanan et al. [56] investigated acetylene gas as fuel in air-cooled single-cylinder direct injection CI engines under variable load scenarios as an alternative to diesel fuel in dual fuel mode. In this case, diesel was employed as the primary fuel and acetylene as the supplementary fuel. The engine performance worsened with this dual fuel, and there was a reduction in smoke, CO, and UHC owing to improved combustion, as well as an increase in NO_x production and peak pressure due to high-speed flame propagation.

Acetylene-diesel dual fuel compression ignition (CI) engines are being investigated as a potential alternative to standard diesel engines. Despite substantial research in this area, some crucial research gaps must be filled to increase our understanding and optimization of these engines. To begin, there are no comprehensive studies that investigate the combined effects of engine load, CR, and acetylene gas flow rate (AGFR) on the performance, combustion characteristics, and emissions of acetylene-diesel dual fuel CI engines in the existing literature. Most studies have focused on single factors, with little systematic investigation of the interaction between several control variables and their overall impact on engine performance. Also, while RSM has been demonstrated to be an effective optimization tool for a variety of engine systems, its application to acetylene-diesel dual fuel CI engines remains limited. The full potential of RSM in this context must be investigated and leveraged to determine the optimal operating conditions for these engines in terms of efficacy, emissions, and overall performance [57]. The major goal of this study work is to fill the gaps described above by performing a thorough examination of the effects of engine load, CR, and AGFR on the performance of acetylene-diesel dual fuel CI engines. It will be achieved by using RSM as a strong optimization tool to determine the ideal operating parameters that result in the best trade-off between engine efficiency, emissions reduction, and overall efficiency. This study also intends to give useful insights into the combustion properties and pollutant production mechanisms in acetylene-diesel dual fuel CI engines, particularly under diverse operating situations. The findings of this study will help to improve our understanding of the underlying mechanisms and will

aid in the design and development of improved and environmentally friendly dual-fuel CI engines. A schematic of the research in this study is depicted in Fig. 1.

2. Materials and methods

2.1. Test setup

In the present investigation, the experiments were carried out on a research engine with a maximum load capacity of 3.5 kW. As shown in Fig. 2, the test engine is naturally aspirated and capable of accepting variations in operating characteristics such as injection timings and loading conditions. The engine was a single-cylinder and four-stroke, and it has provision for loading using an eddy current dynamometer. The engine had provisions for varying the compression ratio. The major specifications of the test engine are described in the authors' prior work. The fuel injection system consists of an injector with three circular holes of 0.3 mm diameter spraying diesel at a 120° angle, a fuel pump made by Bosh, and a fuel filter. In this study, the fuel injection pressure was maintained at 220 bar.

The baseline test was started by operating the engine in diesel mode. Initially, the test engine was run at a lower operating setting i.e., CR = 167.5 and FIT = 23°bTDC for five different levels of engine loadings (20 %, 40 %, 60 %, 80 %, and 100 %). The engine is operated with high-speed diesel as pilot fuel. The acetylene gas was drawn from an industrial-grade gas cylinder using a digital gas regulator. It was ensured that fire extinguishers were available nearby to test setup and proper ventilation was ensured. In addition, the main properties of the test fuel are listed in Table 1.

2.2. Design of experiments with Response Surface Methodology

Design of experiments (DoE) is a statistical strategy for methodically organizing, carrying out, and analyzing experiments with the objective to optimize and enhance procedures, goods, or systems. It entails carefully modifying a system's input variables (factors) to evaluate their impacts on the desired output response (outcome). DoE is extensively utilized in many industries, including engineering, production, chemistry, and social sciences, to study and comprehend the link between factors and responses in an efficient and effective manner. Response Surface Methodology (RSM) is a subset of DoE that focuses on developing mathematical models to characterize the link between the input factors and the response, it could be used to optimize the experimental results [58,59]. It attempts to identify the ideal settings for the components that will deliver the best potential response. Whenever the connection between variables and response is intricate and non-linear, RSM is very beneficial [60]. RSM represents the answer as a function of the components using polynomial regression models. A second-order polynomial regression model with two components (x_1 and x_2) has the following general form [61,62]:

$$Y = \beta_0 + \beta_1x_1 + \beta_2x_2 + \beta_{11}x_1^2 + \beta_{22}x_2^2 + \beta_{12}x_1x_2 + \varepsilon \tag{1}$$

Wherein, the anticipated response of the variable (output) is denoted by Y. The regression coefficients that were determined based on the data are $\beta_0, \beta_1, \beta_2, \beta_{11}, \beta_{12},$ and β_{22} . x_1 and x_2 are the levels (values) of the two elements under consideration. ε is the error term that represents the amount of residual variation that the model does not explain. RSM's purpose is to calculate the coefficients ($\beta_0, \beta_1, \beta_2, \beta_{11}, \beta_{12},$ and β_{22}) from experimental data and subsequently, employ the model to improve the response by determining which factor values maximize or minimize it. RSM commonly utilizes several experimental designs, like Central Composite Design (CCD) or

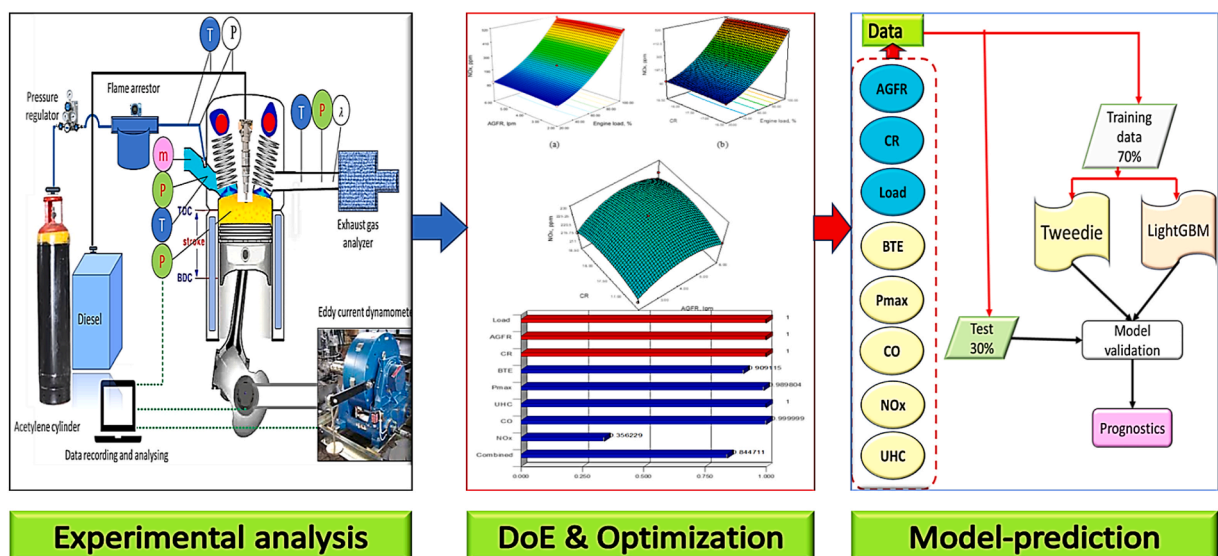


Fig. 1. Schematic diagram of the research.

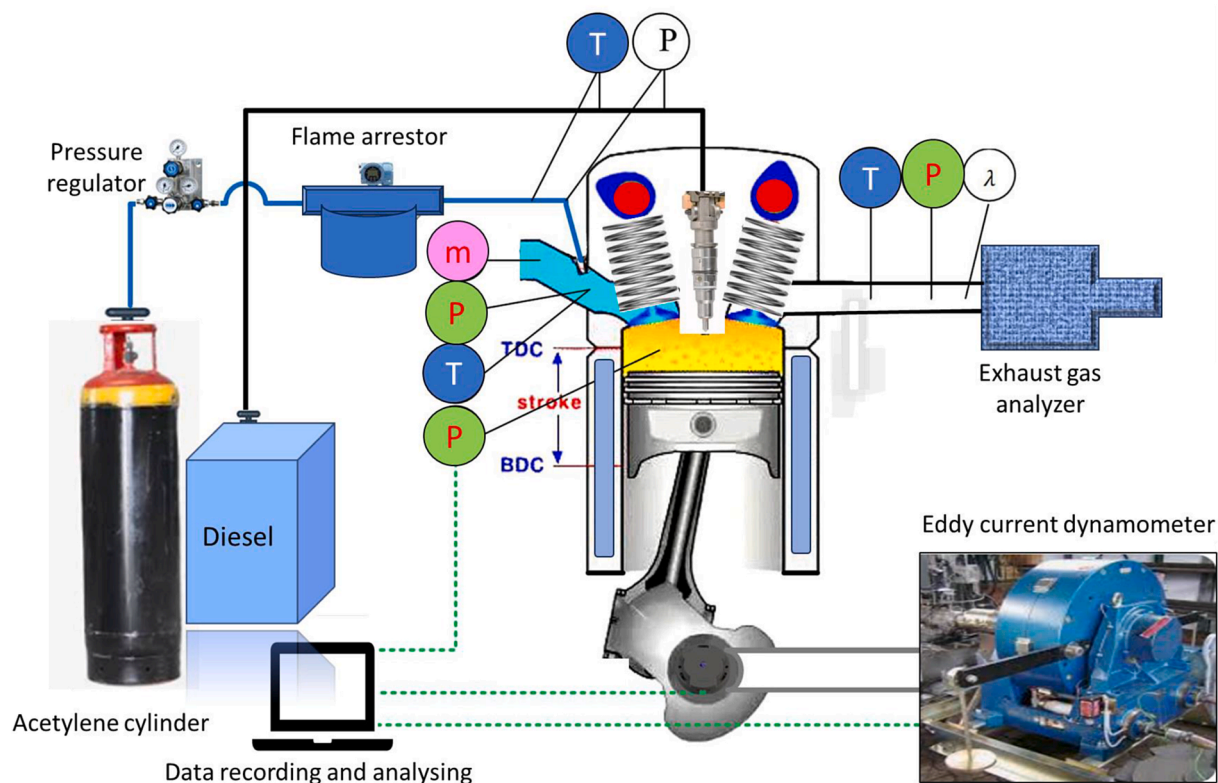


Fig. 2. Employed engine test setup.

Table 1

Main characteristics of test fuel.

Properties	Diesel	Acetylene
Chemical formulae	$C_{12}H_{26}$	C_2H_2
Density (@ 1.01325 bar and 293 K)	857 kg/m^3	1.092 kg/m^3
Lower calorific value (kJ/kg)	42600	48300
Stoichiometric air-fuel ratio (kg/kg)	14.5	13.2
Cetane number	48	–
Viscosity	2.88 cSt	–

Box-Behnken Design, to do this. These designs allow for efficient investigation of the factor space whilst reducing the number of tests necessary [62].

These designs have been selected for their orthogonality as well as rotatability qualities, which aid in predicting the coefficients with high precision and facilitate simple viewing of the response surface. RSM's mathematical methodology includes regression analysis for calculating coefficients and statistical tests to determine the relevance of components. The model is then utilized to discover ideal factor values (to maximize or decrease the response) by accounting for crucial points such as stationary points and employing numerical optimization methods. Overall, RSM is a strong and effective approach that integrates experimental design, statistical evaluation, and mathematical modeling to improve processes and systems, reducing time and resources while meeting performance goals.

2.3. Uncertainty analysis

For the uncertainty analysis associated with any experimental investigation, the broadly established Perturbation techniques are applied [63]. Fig. 3 shows the relative errors and uncertainty associated with various performance metrics. In the case of peak pressure measurement, the relative error (RE) was only 0.05 %, and RE was 1.2 % for engine speed. In the case of engine load RE was 0.2 % while it was 1 % for the lower calorific value (LCV) of the fuel. In the case of emission measurement, it was altogether 2.65 %. These disparities underline the need for precise measurement and calibration methodologies in engine performance evaluation. On the other hand, The AGFR uncertainty was estimated as 1.1 %, it was 1.2 % for air flow rate. Uncertainty was 1.6 % and 1.3 % for BTE and brake power respectively.

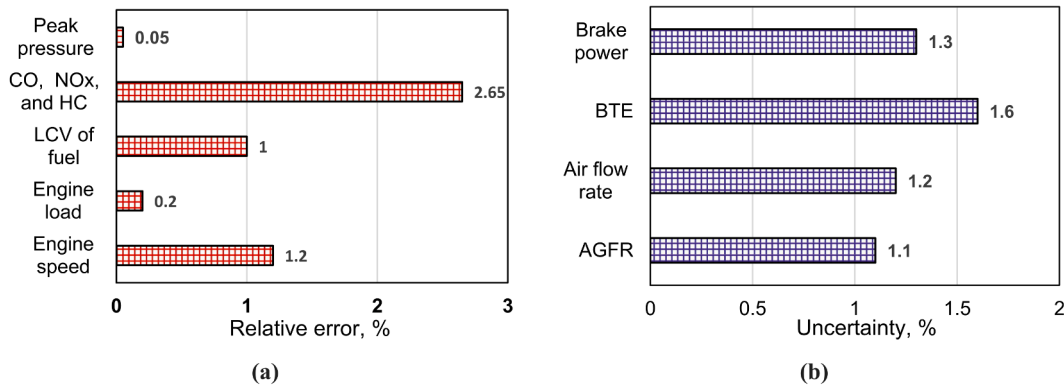


Fig. 3. (a) Relative error (b) Uncertainty.

2.4. Design matrix

The RSM-based Design of Experiment (DoE) was used to formulate an experiment strategy for conducting the experiments as shown in Table 2. The Box-Behnken design was used in the present study. Box-Behnken Design, an intriguing variation of RSM, offers a novel way to experimental design. This strategy enables researchers to investigate three variables at the same time, making it perfect for optimizing answers with minimal resources. Box-Behnken Design quickly estimates primary impacts, quadratic effects, and two-way relationships by integrating fewer runs while keeping the design's orthogonality [64]. Its rotatability enables researchers to investigate response surfaces from various angles, aiding the identification of optimal factor values. Box-Behnken Design is a tempting choice for an array of research and manufacturing processes, where it reveals hidden insights and accelerates breakthroughs in numerous sectors, thanks to its ability to reduce experimental error and deliver exact model predictions.

2.5. Analysis of variance

Analysis of variation (ANOVA) is a method of statistics that evaluates the variation across groups to help uncover the secrets hidden inside data. ANOVA has evolved as a basic tool in many domains such as psychology, biology, engineering, and economics [58,65,66]. The core of ANOVA is its capacity for comparing means and examining the influence of various factors on data variance. ANOVA allows researchers to identify whether or not differences between groups are significant by splitting total variability into separate sources [67]. The ANOVA outcomes for BTE and Pmax are listed in Table 3, while for emission data is listed in Table 4.

2.6. Prognostics analysis with modern machine learning

2.6.1. Tweedie

Incorporating the statistical characteristics of the Tweedie distribution, the Tweedie machine learning approach adds a unique dimension to predictive modelling [68]. When represented mathematically:

Table 2
Design matrix for experiment.

Run	Control factors			Response variables				
	Load, %	AGFR, lpm	CR	BTE, %	Pmax, bar	UHC, ppm	CO, ppm	NOx, ppm
3	60	2	18.5	21.8	69.8	71	111	215
9	100	2	17.5	27.4	74.9	71	112	511
12	20	2	17.5	11.5	46	100	137	86
15	60	2	16.5	21.1	68.4	72	145	211
1	100	4	18.5	28.7	76.9	64	102	506
2	20	4	18.5	11.3	46.5	92	108	96
4	60	4	17.5	23.4	71.3	59	112	229
6	60	4	17.5	23.4	71.3	59	112	229
7	60	4	17.5	23.4	71.3	59	112	229
10	20	4	16.5	11.3	46	96	128	94
11	60	4	17.5	23.4	71.3	59	112	229
13	60	4	17.5	23.4	71.3	59	112	229
17	100	4	16.5	27	76	66	124	507
5	60	6	18.5	22.5	69.4	64	106	224
8	60	6	16.5	21.7	68.7	67	124	218
14	20	6	17.5	11.1	45.4	93	119	94
16	100	6	17.5	28.4	75.4	64	120	506

Table 3
ANOVA outcomes for engine performance.

Source	BTE			Pmax		
	SS	Value of 'F'	p' value (Prob > F)	SS	Value of 'F'	p' value (Prob > F)
Model	598.72	6898.87	< 0.0001	2169.89	4927.56	< 0.0001
Model	549.46	56981.17	< 0.0001	1779.06	36360.38	< 0.0001
P-Load	0.45	46.80	0.0002	0.01	0.10	0.7585
Q-AGFR	1.28	132.74	< 0.0001	1.53	31.30	0.0008
R-CR	0.49	50.81	0.0002	0.30	6.18	0.0418
PQ	0.72	74.93	< 0.0001	0.04	0.82	0.396
PR	0.00	0.26	0.6263	0.12	2.50	0.1576
QR	37.89	3929.82	< 0.0001	364.17	7442.86	< 0.0001
P ²	2.69	279.45	< 0.0001	10.44	213.47	< 0.0001
Q ²	2.87	297.19	< 0.0001	1.78	36.36	0.0005
Residual	0.07			0.34		
Lack of Fit	0.07			0.34		
Pure Error	0.00			0.00		
Cor Total	598.79			2170.23		

Table 4
ANOVA outcomes for emission data.

Source	UHC			CO			NOx		
	SS	Value of 'F'	p' value (Prob > F)	SS	Value of 'F'	p' value (Prob > F)	SS	Value of 'F'	p' value (Prob > F)
Model	3295.24	2562.96	< 0.0001	1957.118	22.06	0.0002	369558	5155.77	< 0.0001
P-Load	1682.00	11774.00	< 0.0001	144.50	14.66	0.006	344450	43249.33	< 0.0001
Q-AGFR	84.50	591.50	< 0.0001	162.00	16.43	0.0048	45.13	5.67	0.0489
R-CR	12.50	87.50	< 0.0001	1104.50	112.05	< 0.0001	15.13	1.90	0.2106
PQ	0.00	0.00	1	169.00	17.14	0.004	42.25	5.30	0.0547
PR	1.00	7.00	0.0331	1.00	0.10	0.759	2.25	0.28	0.6115
QR	1.00	7.00	0.0331	64.00	6.49	0.038	1.00	0.13	0.7335
P ²	1216.84	8517.89	< 0.0001	16.84	1.71	0.232	24964.2	3134.52	< 0.0001
Q ²	151.58	1061.05	< 0.0001	269.47	27.34	0.001	191.84	24.09	0.0017
R ²	51.58	361.05	< 0.0001	9.47	0.96	0.359	116.05	14.57	0.0066
Residual	1.00			69			55.75		
Lack of Fit	1.00			69			55.75		
Pure Error	0.00			0			0		
Cor Total	3296.24			2026.118			369614		

$$y_i = \mu_i + \Phi y_i^{(p-1)} - \gamma y_i^{(q-1)} + \epsilon_i \tag{2}$$

In Eq. (2), the y_i denotes response variable, μ_i shows mean value, Φ shows the dispersion parameter, and γ also denotes the dispersion parameter. This formulation incorporates the interaction that exists between the response's mean and variance in addition to an extensive range of data kinds, such as continuous, number, and sparse data. The Tweedie ML technique can handle complicated datasets and provide reliable predictions because of its versatile mathematical underpinnings, particularly in situations when conventional models fall short. This ground-breaking approach opens new doors for enhanced prediction capabilities and useful insights across a variety of disciplines by seamlessly incorporating statistical insights into ML frameworks [69,70]. Due to its flexibility in handling multiple data types and complex variance frameworks, the Tweedie ML method, which has its roots in its original mathematical formulation, finds use in many different fields. Similar to how it helps in modelling over-dispersed species abundance data in ecology. It is appropriate for examining consumer records of transactions or social media interactions since its incorporation into predictive analytics enhances the precision of forecasts in settings characterized by diverse data sources [71,72].

The Tweedie machine learning strategy does, however, have some drawbacks, just like any other technique. When the fundamental information considerably departs from these assumptions, it may produce inferior results due to its reliance on the Tweedie distribution assumptions. In addition, rigorous calibration is needed to establish the optimum values for the dispersion parameters (and), which could prove difficult in practice. Despite these drawbacks, the method's distinctive integration of distribution features into ML opens the door for more precise and nuanced modelling across a variety of domains, bridging the gap between statistical insights and based data predictions.

2.6.2. LighGBM

The LightGBM ML approach offers a solid mathematical basis for predictive modeling and is renowned for its effective gradient-boosting architecture. At its core, LightGBM minimizes loss while iteratively incorporating weak learners to optimize an objective function. The weighted forecasts of all weak learners are added up to create the final prediction, with the weights being based on how

much each weak learner contributed to reducing the loss [73]. In the case for a regression objective having N training samples consisting of M features. Here the loss function is defined as following if y_i and \hat{y}_i , are target value and predicted value, respectively.

$$\text{Loss function} = (y_i, \hat{y}_i) \quad (3)$$

In case of each iteration 't', the loss function's negative gradient comparative to previous model's predictions is estimated as:

$$g_i^{(t)} = -\frac{\partial L(y_i, \hat{y}_i^{t-1})}{\partial \hat{y}_i^{t-1}} \quad (4)$$

The construction of tree in case of LightGBM grows in leafwise manner. At each node 'm' of the tree the optimized split is such a way to minimize the loss function. In this case if 'S' represents sample set reaching leaf node 'm'. The optimized point of split is found by:

$$\text{split}_m = \arg \min_{\text{split}} \sum_{i \in S} L(y_i, \hat{y}_i^{(t-1)} + \text{split}) \quad (5)$$

On the tree construction completion, the output value in respect of each leaf node m is estimated using a weighted sum of negative gradient of the sample in the leaf:

$$\text{leaf_output}_m = -\frac{\sum_{i \in S} g_i^{(t)}}{\sum_{i \in S} h_i^{(t)} + \lambda} \quad (6)$$

This presents a simplified overview of the LightGBM process which employs iteratively fitting trees to the negative gradients of the loss function and updating the model's predictions consequently.

The leaf-wise tree growth approach of the method adds more mathematical sophistication. In order to generate more informative splits, it chooses the leaf with the highest delta loss to expand. By adding regularisation terms in its objective function, the LightGBM method also addresses overfitting by balancing the trade-off between bias and variance. With its histogram-based methodology, continuous characteristics are quantized into discrete bins, which enables quick computations and graceful handling of missing values [73,74]. Due to its high computational efficiency, LightGBM is well-suited for real-time or huge dataset applications. Its depth-first growth method, though, might make it more sensitive to outliers. Additionally, parameter adjustment, which includes learning rate and number of leaves, calls for careful thought. However, LightGBM's role as a high-performance tool for diverse ML tasks, from classification to regression, and from ranking to recommendation algorithms, is supported by its use of mathematical elegance and optimization approaches [75,76].

3. Results and discussion

3.1. Correlation among data

The data values gathered through extensive lab-based testing were checked for correlation to reveal statistical linkage among data columns. It was observed that a high correlation (0.96) exists between engine load and BTE, and 0.97 between engine load and NOx emission. This shows that at higher engine loads higher BTE and NOx are observed. On the other side, the acetylene gas flow rate shows a slightly positive effect on BTE while it shows negative effects on UHC and CO emission. This is attributed to the fact that in a constant-speed engine, the fuel supply is controlled with a mechanical governor fitted on the engine. At higher loads, to maintain the engine speed constant, a higher amount of fuel is supplied to the combustion chamber [77,78]. This results in higher temperature and pressure in the combustion chamber resulting in improved BTE and higher generation NOx [79–81]. Similarly, CR shows positive effects on BTE peak pressure and NOx. The low values of correlation are caused by quadratic terms since the effects of AGFR and CR are not linear. The correlation plot is depicted in Fig. 4.

3.2. Brake thermal efficiency

The F-value of the analysis, with a value of 6898.87, suggests that the predictive value is very significant. The possibility of seeing a "Model F-value" of this size owing to noise is a mere 0.01 %. This implies that the model is not an outcome of random chance, but rather a meaningful representation of the data's inherent relationships. Furthermore, the values of "Prob > F" for different model terms are investigated in order to establish their importance. When "Prob > F" is less than 0.0500, it indicates that the model terms are statistically noteworthy. The words A, B, C, AB, AC, A², B², and C² are all relevant model terms in this scenario. Values larger than 0.1000, on the contrary, imply that the predictive terms are not significant. If there are multiple inconsequential model terms, except those essential for sustaining hierarchy, reducing them may be advantageous to enhance model performance. The computed standard deviation is 0.098, while the R-squared value is 0.9999, suggesting that the model accounts for a significant amount of the overall variation in the data. Furthermore, the Adjusted R-squared score of 0.9997 validates this conclusion and suggests that the model fits the data well. The Coefficient of Variation (C.V.%) is 0.46, indicating that the relative volatility of the model is fairly low. The Predicted R-squared value of 0.9982 agrees with the Adjusted R-squared number of 0.9997, confirming that the model's predictive capability is adequate. The BTE model can be expressed as:

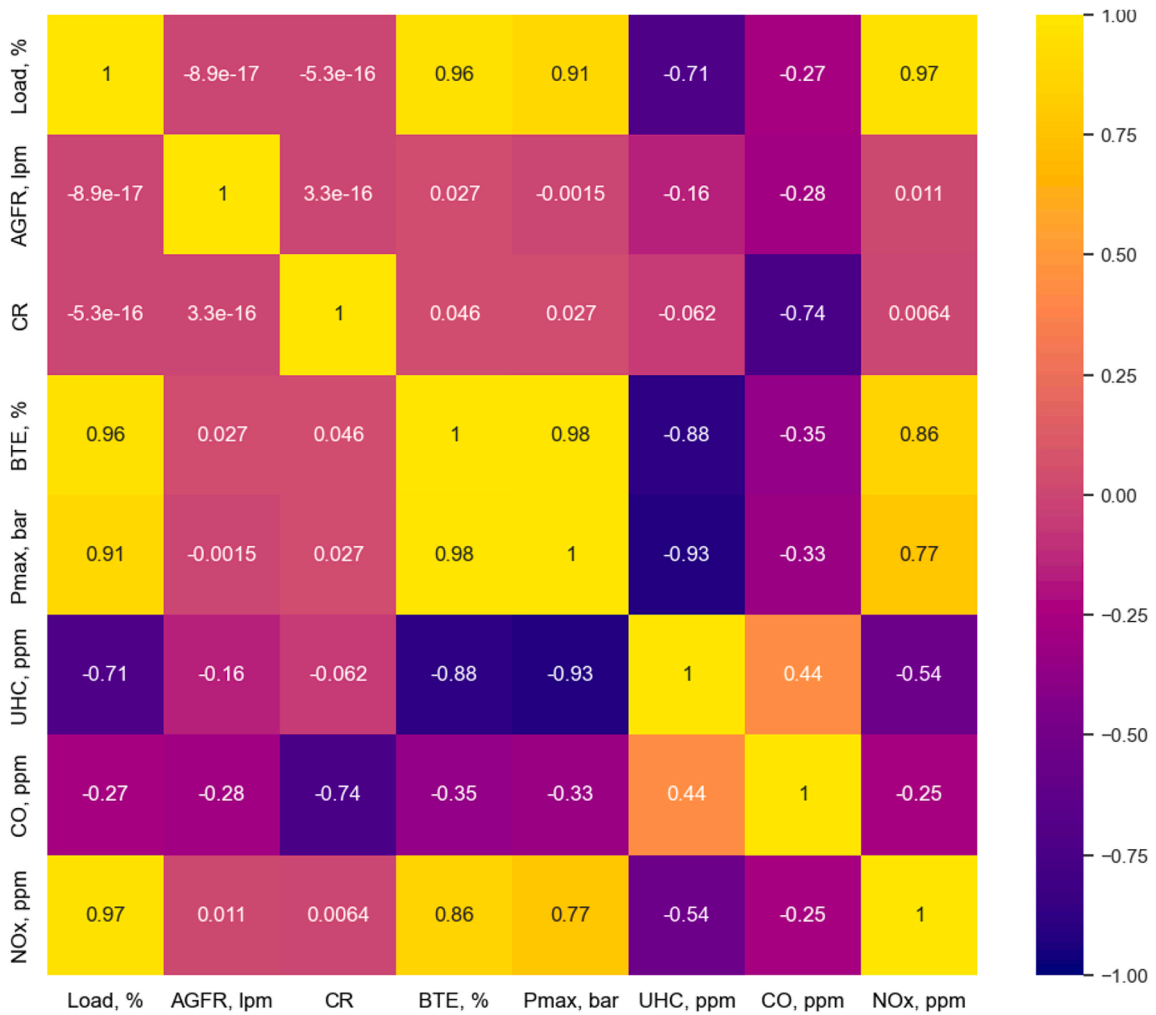


Fig. 4. Correlation plot.

$$\begin{aligned}
 \text{BTE} = & -246.03 + 0.23 P + 1.24 Q + 28.59 R + 0.0044 PQ \\
 & + 0.011 PR + 0.0125 QR - 0.0018 P^2 - 0.2 \hat{Q}^2 - 0.825 \hat{R}^2
 \end{aligned}
 \tag{2}$$

Herein, P denotes the acetylene gas flow rate, Q represents engine load, and R denotes the compression ratio of the test engine.

Adequacy Precision, which evaluates the signal-to-noise ratio, is an important metric for determining model dependability. A ratio above four is regarded as ideal, and the Adequacy Precision in this example is 232.193, suggesting a high signal-to-noise ratio. The finding adds to the model's usefulness for exploring the design space. In summary, statistical analysis of parameters shows that the model is very significant and valid for the supplied data. The model has strong predictive power and dependability, making it a helpful tool for exploring the design space and making educated decisions based on the variables' correlations. The surface diagram for BTE is depicted in Fig. 5.

As engine load increases, so does the power demand, culminating in increasing fuel consumption along with higher combustion temperatures. The engine performs closer to its intended capacity under raised loads, resulting in better thermal efficiency, as shown in Fig. 5a and b. Excessive loads, on the contrary, can increase mechanical losses and decrease thermal efficiency owing to friction and other inefficiencies [82,83]. In a diesel engine, using acetylene gas as an alternative fuel can change the combustion parameters. Acetylene gas has a greater flame speed along with energy content than diesel, allowing for faster and more efficient burning [84]. When the acetylene flow rate is modified, it can result in greater fuel-air mixing and more thorough combustion, which improves the thermal efficiency of the brakes [85], as depicted in Fig. 5c. Throughout the compression stroke, a larger compression ratio generates increased air temperature and pressure. This, in turn, improves combustion efficiency by compressing and heating the air-fuel combination, allowing for greater energy delivery throughout the power stroke and increased BTE [86–88].

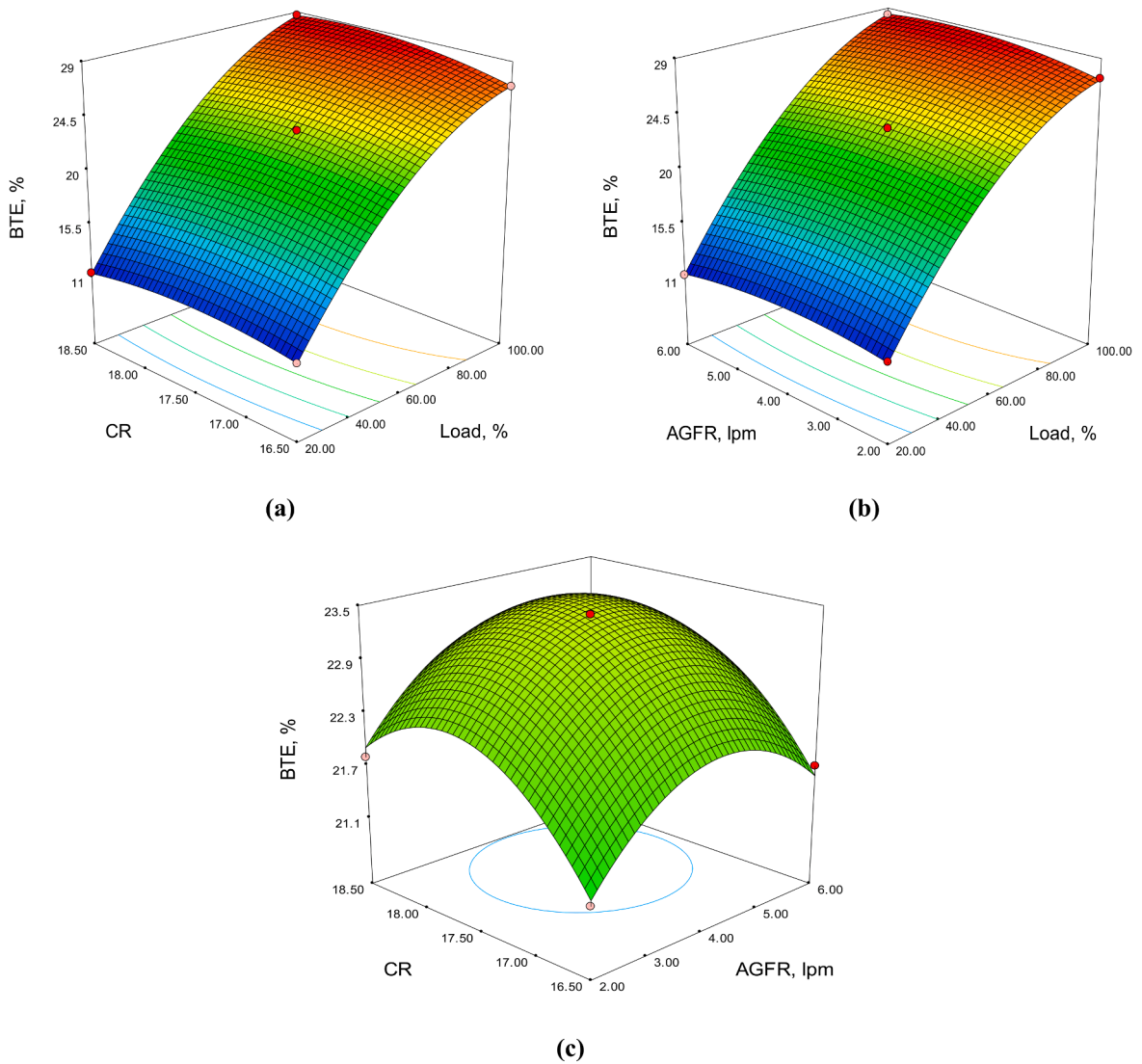


Fig. 5. Surface diagrams for BTE depicting the effects of (a) CR and engine load; (b) Acetylene gas flow rate and engine load; (c) CR and Acetylene gas flow rate.

3.3. Peak cylinder pressure

The model for P_{max} developed using ANOVA is given in Eq. (3). The exceptional “Model F-value” of 4927.56 emphasizes the model’s relevance, meaning that such an as big “Model F-Value” is extremely unlikely to arise due to random noise. This implies that the model incorporates significant correlations between the variables. A closer look at the “Prob > F” values indicates that numerous model variables, including A, C, AB, A^2 , B^2 , and C^2 , are considered important since their respective probability are less than 0.0500. These key model elements are critical in explaining the variances found in the data, confirming their relevance in the overall model.

$$\begin{aligned}
 P_{max} = & -187.64 + 1.01 P + 4.46 Q + 23.39 R + 0.0034 PQ + 0.0025 PR \\
 & - 0.088 QR - 0.0058 \hat{P}^2 - 0.394 \hat{Q}^2 - 0.65 \hat{R}^2
 \end{aligned}
 \tag{3}$$

Model terms with a probability larger than 0.1000, on the other hand, are deemed insignificant. The Forecast R-squared value of 0.9982 corresponds in magnitude with the Adjusted R-squared number of 0.9997, confirming that the model’s ability to predict is adequate. Adequacy Precision, which evaluates the signal-to-noise ratio, is an important metric used to assess model dependability. A ratio greater than 4 is regarded as ideal, and the Adequacy Precision in this example is 232.193, demonstrating a high signal-to-noise ratio. The finding adds to the model’s usefulness for exploring the design space. In summary, statistical parameter analysis shows that the model is very significant and valid for the supplied data. The model has strong predictive power and dependability, making it a helpful tool for investigating the design space and making educated decisions based on the variables’ correlations.

More fuel is introduced into the cylinder as the engine load rises to match the increasing demand for power. As a consequence, more fuel-air mixture is compressed in the cylinder throughout the compression stroke [89]. As a result, compressing an increased amount of mixture leads to higher peak pressures throughout the combustion process, as shown in Fig. 6a. Because acetylene gas is extremely reactive, it accelerates and completes the combustion process of the fuel-air combination [90,91]. When the flow rate of acetylene gas is raised, it introduces more energy into the combustion process, enabling the combustion to proceed more quickly and furiously. As a result, the rate of pressure rise throughout combustion accelerates, resulting in larger cylinder peak pressures, as shown in Fig. 6b. A greater pressure at the final stage of the compression stroke results from a more efficient compression process. Higher compression ratio results in higher cylinder pressures (Fig. 6c), including the peak pressure following combustion [92,93].

3.4. Unburnt hydrocarbon emission

The model possesses an F-value of 2562.96, demonstrating that it is extremely significant. The likelihood of such a significant "Model F-Value" being influenced by noise is under 0.01 %, showing a strong link between the variables. Model terms are of statistical significance if "Prob > F" is less than 0.0500. Model terms A, B, C, AC, BC, A², B², and C² are all important in this scenario, indicating that they have a substantial impact on the model's output. Values larger than 0.1000 for "Prob > F" indicate that the relevant

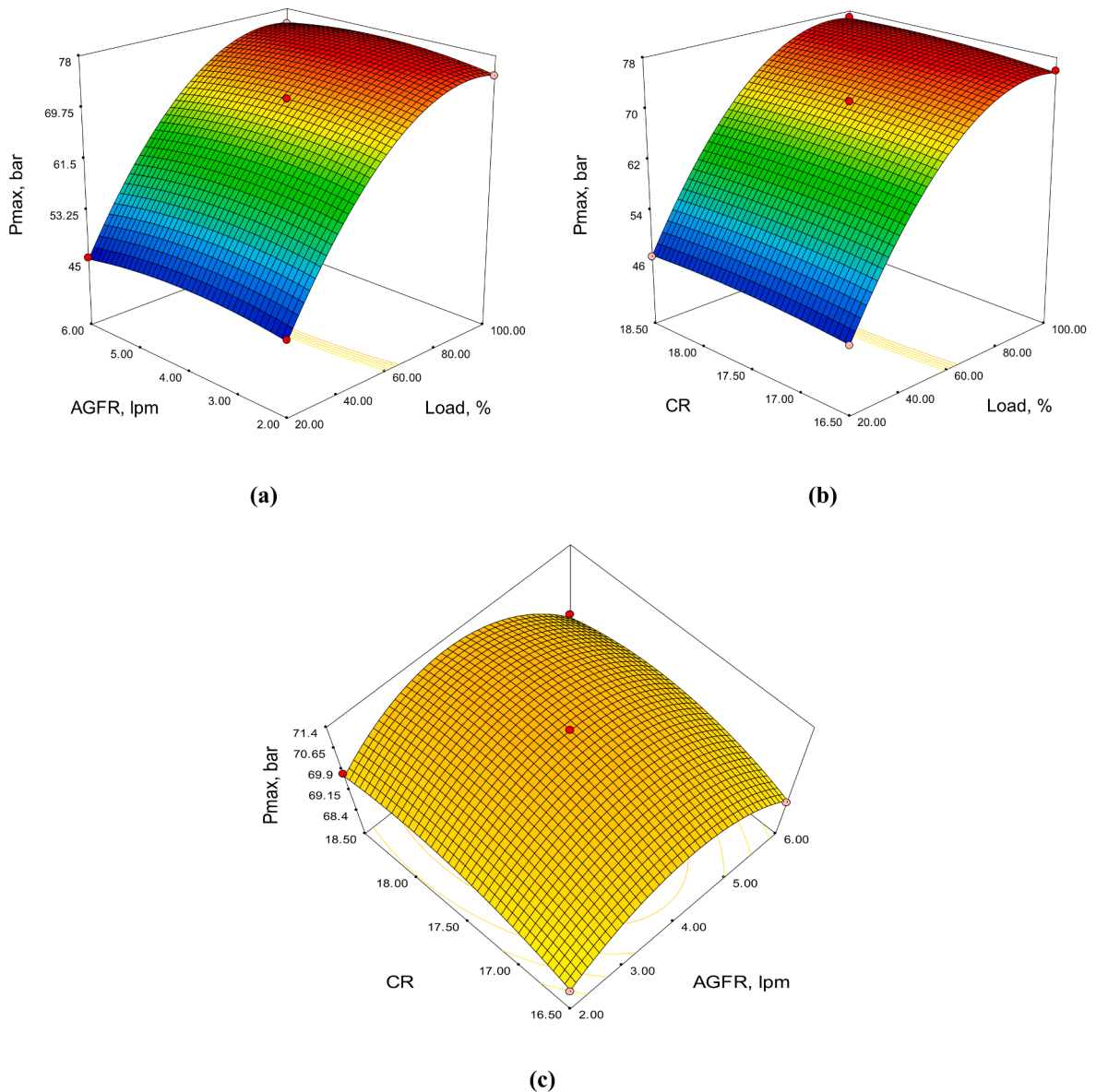


Fig. 6. Surface diagrams for Pmax depicting the effects of (a) CR and engine load; (b) Acetylene gas flow rate and engine load; (c) CR and Acetylene gas flow rate.

model terms are not significant. Reducing model complexity may result in benefits if there are multiple inconsequential model terms (excluding those necessary for hierarchy). The UHC model can be expressed as Eq. (4):

$$UHC = 1238.88 - 1.86 P - 9.25 Q - 123.5 R + 0.0125 PR - 0.25 QR + 0.01P^2 + 1.5 \hat{Q}^2 + 3.5 \hat{R}^2 \quad (4)$$

The model performs well, having an R-squared value of 0.9997 suggesting that it explains nearly all of the variation in the data. This beneficial degree of goodness of fit is reinforced by the Adjusted R-squared of 0.9993. The coefficient of variation is 0.53, indicating that the data is not variable. The “Pred R²” score of 0.9951 corresponds well with the “Adj R²,” demonstrating the model's dependability.

Engine load, AGFR, and CR are all three essential elements that influence the total amount of UHC emissions released by a diesel engine. The larger loads often result in greater-intensity combustion processes. At higher loads, the engine's cylinders endure higher temperatures and pressures, resulting in more complete fuel combustion and, as a result, lower UHC emissions [94]. Indeed, it could be seen the relationship between UHC emissions with compression ratio and engine loads in Fig. 7a. When employed as an additive, acetylene gas improves combustion efficiency by accelerating the ignition and burning of any leftover fuel particles which could be contributing to unburned hydrocarbons, as shown in Fig. 7b [95,96]. It works by encouraging more complete combustion of diesel fuel. Finally, compression flow, as indicated by the compression ratio, is critical. A greater CR causes higher compression tempera-

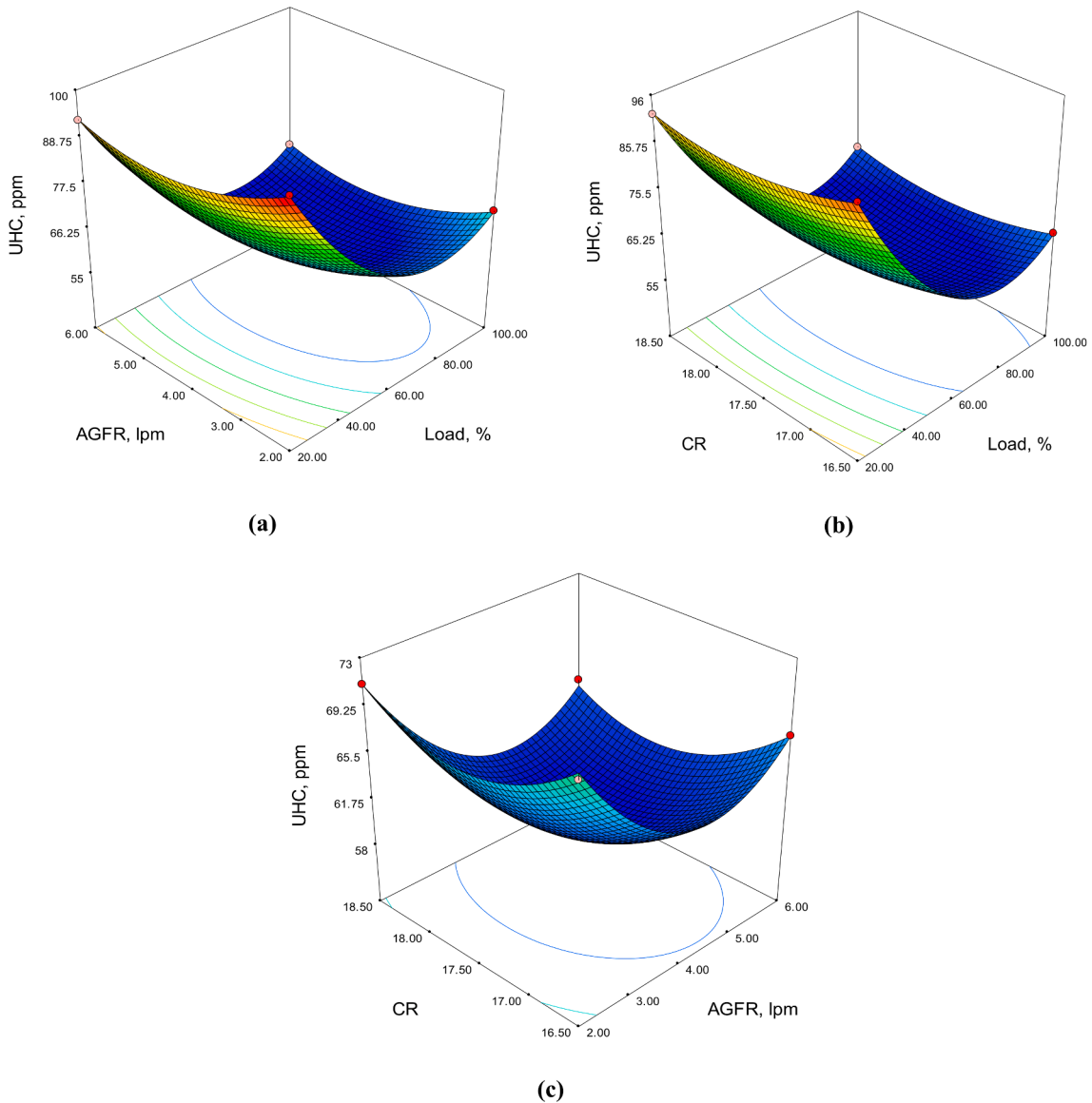


Fig. 7. Surface diagrams for UHC emission depicting the effects of (a) CR and engine load; (b) Acetylene gas flow rate and engine load; (c) CR and Acetylene gas flow rate.

tures and pressures, therefore improving combustion efficiency, and reducing unburned hydrocarbon production, as shown in Fig. 7c. Finding a satisfying medium between these variables is critical for optimizing engine performance, fuel consumption, and environmental effects, resulting in cleaner and more ecological diesel engine running [97,98].

3.5. Carbon mono-oxide emission

The Model F-value of 22.06 demonstrates that the model is very significant, and it has a 0.02 % chance of attaining such a huge F-value as a result of noise. Model terms have significance when “Prob > F” is less than 0.0500. The words A, B, C, AB, BC, and B² are all pertinent in this scenario. Values larger than 0.1000, on the other hand, indicate that the model terms are not significant. Reduce the model if it contains multiple inconsequential model terms. The standard deviation of 3.14, in addition to an R-squared value of 0.9659, indicates that the model describes a significant amount of variability in the data. This inference is supported by the adjusted R-squared value of 0.9222, which indicates a satisfactory fit for the model. The term “Adeq Precision” refers to the signal-to-noise ratio. A ratio larger than 4 is preferred, and the ratio of 17.754 shows a suitably strong signal in this circumstance. As a consequence, this model may be employed to reliably investigate the design space. The model developed for CO emission for the present study can be expressed as:

$$CO = 975.25 - 0.36 P - 58.13 Q - 71.5 R + 0.08 PQ - 0.0125 PR + 2 QR + 0.00125 \hat{P}^2 + 2 \hat{Q}^2 + 1.5 \hat{R}^2 \quad (5)$$

In a CI engine, raising the load may result in more CO emissions. Whenever the engine is running at greater loads, it takes more fuel to create the necessary power, resulting in higher diesel fuel combustion [99,100]. This increased combustion might result in incomplete combustion, which produces more CO as a byproduct. When acetylene gas is put into the engine in regulated proportions, it can improve combustion and minimize CO emissions, as depicted in Fig. 8a and b. The inclusion of acetylene gas in the air-fuel combination can increase combustion efficiency, resulting in more complete burning of diesel fuel and, as a result, fewer CO emissions [101,102]. Higher compression ratios result in higher compression stroke temperatures and pressures, resulting in better combustion, as depicted in Fig. 8c. There is a reduced amount of unburned fuel with enhanced combustion, which equals lower CO emissions. A lower compression ratio, on the contrary, may result in incomplete combustion and increased CO emissions.

3.6. Oxides of nitrogen

The model's relevance and efficacy are revealed via analysis. The model's F value of 5155.77 suggests that the model is very substantial, with only a 0.01 % probability of such a huge F-value being caused by noise. In addition, values less than 0.0500 for “Prob > F” suggest that model elements A, B, A², B², and C² are statistically significant. Values larger than 0.1000, on the other hand, imply that some model terms are not relevant. The model's ability to explain variance in the data has been shown by the standard deviation of 2.82 and R-squared value of 0.9998. Furthermore, the mean of 259.59 and Adjusted R-squared value of 0.9997 reflect the model's great goodness of fit. The mathematical model for NOx emission is expressed as:

$$NOx = -1584.56 - 0.097 P + 12.75 Q + 185.25 R - 0.04 PQ - 0.019 PR + 0.25 QR + 0.048 \hat{P}^2 - 1.69 \hat{Q}^2 - 5.25 \hat{R}^2 \quad (6)$$

The model's forecasts are in high agreement with the actual data, as demonstrated by the Coefficient of Variation (C.V.) of 1.09 % and the Forecast R-squared value of 0.9976. The model's accuracy in forecasting is measured by the “PRESS” value of 892.00, and the Adequate Precision score of 195.315 shows an adequate signal-to-noise ratio. Finally, the research encourages the model's importance and its ability to predict events with high precision. The “Pred R²” of 0.9976 corresponds precisely with the “Adj R²” of 0.9997, adding to the model's dependability. With a reasonable signal-to-noise ratio, this model can be used with confidence to navigate the design space.

To match the greater torque needs, additional fuel is pumped into the combustion chamber as the engine load increases. During the combustion process, this results in increased pressures and temperatures. Because of the oxidation of nitrogen in the air at elevated temperatures, the generation of NOx increases substantially [103,104]. As a result, larger engine loads tend to generate higher NOx emissions due to the increase in temperature in the combustion process [105,106]. Indeed, it could be seen the relationship between NOx emissions with compression ratio and engine loads as using acetylene in from Fig. 9a. When acetylene enters the engine, it combines with the nitrogen oxides, changing them to nitrogen and carbon dioxide, which are not as hazardous to the environment. As a result, increasing the flow rate of acetylene gas can reduce NOx emissions, as shown in Fig. 9b. Higher CR contributes to higher compression stroke temperatures along with pressures, resulting in improved combustion [107]. Normally, due to the greater combustion temperatures, it may also increase the generation of NOx in the combustion chamber [108]. A lower CR, on the other hand, may result in reduced NOx emissions but may affect engine efficiency and power production, as shown in Fig. 9c.

3.7. Desirability-based optimization

Desirability-based optimization is an effective approach for optimizing numerous responses in a process or mechanism at the same time [109,110]. Researchers may use desirability-based optimization to simultaneously enhance BTE, Pmax, CO, UHC, and NOx emissions in the context of the diesel engine's efficiency and emissions. In desirability-based optimization, the first step is to determine the goal values or ranges for each answer. For example, we would wish to optimize BTE and peak pressure whilst reducing CO, HC, and NOx emissions. Each response will be assigned a desirability function that measures how near it is to the goal value. Then, based on the proximity to the goal, we assign distinct desirability scores to each response. A score of one suggests the reaction reaches or ex-

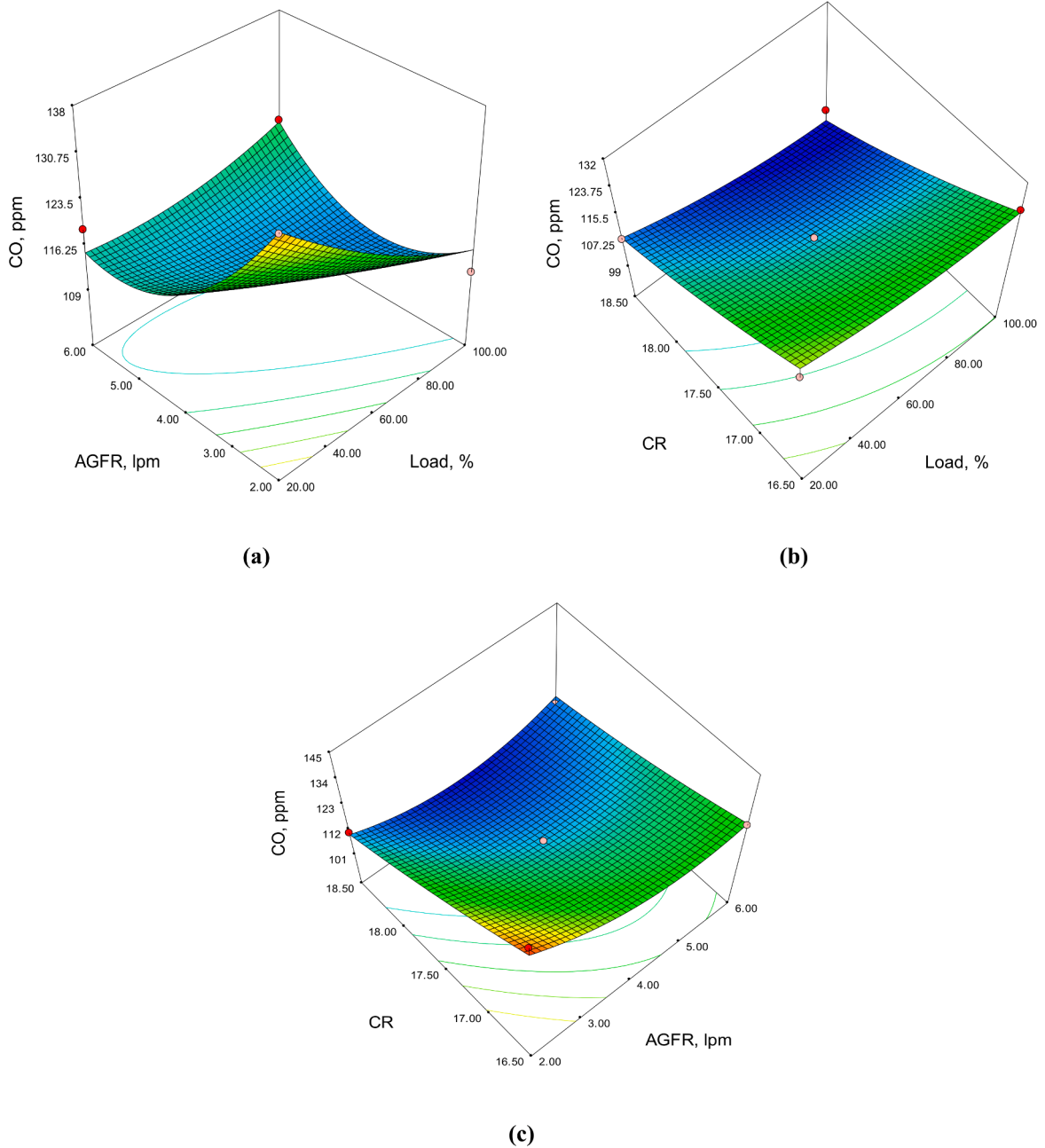


Fig. 8. Surface diagrams for CO emission depicting the effects of (a) CR and engine load; (b) Acetylene gas flow rate and engine load; (c) CR and Acetylene gas flow rate.

ceeds the intended value, whereas a value of zero indicates that it is far from it. Intervals between 0 and 1 denote partial attractiveness. The geometric mean of individual desirability for all responses is used to establish the overall desirability for the engine's efficiency and emissions. The geometric mean method assures that all replies are equally weighted and that no single response dominates the optimization [111,112].

Researchers may use techniques for optimization like the Genetic Algorithm or Particle Swarm Optimization to find the optimal combination of engine operating parameters (engine load, acetylene gas flow rate, and compression ratio). This will provide an ideal set of engine operating settings that maximize BTE and peak pressure while reducing CO, HC, and NOx emissions. Desirability-based optimization offers an in-depth and equitable method for determining the best trade-offs between different options. It enables researchers and engineers to find the optimum acceptable solution for a difficult problem, such as diesel engine performance and emis-

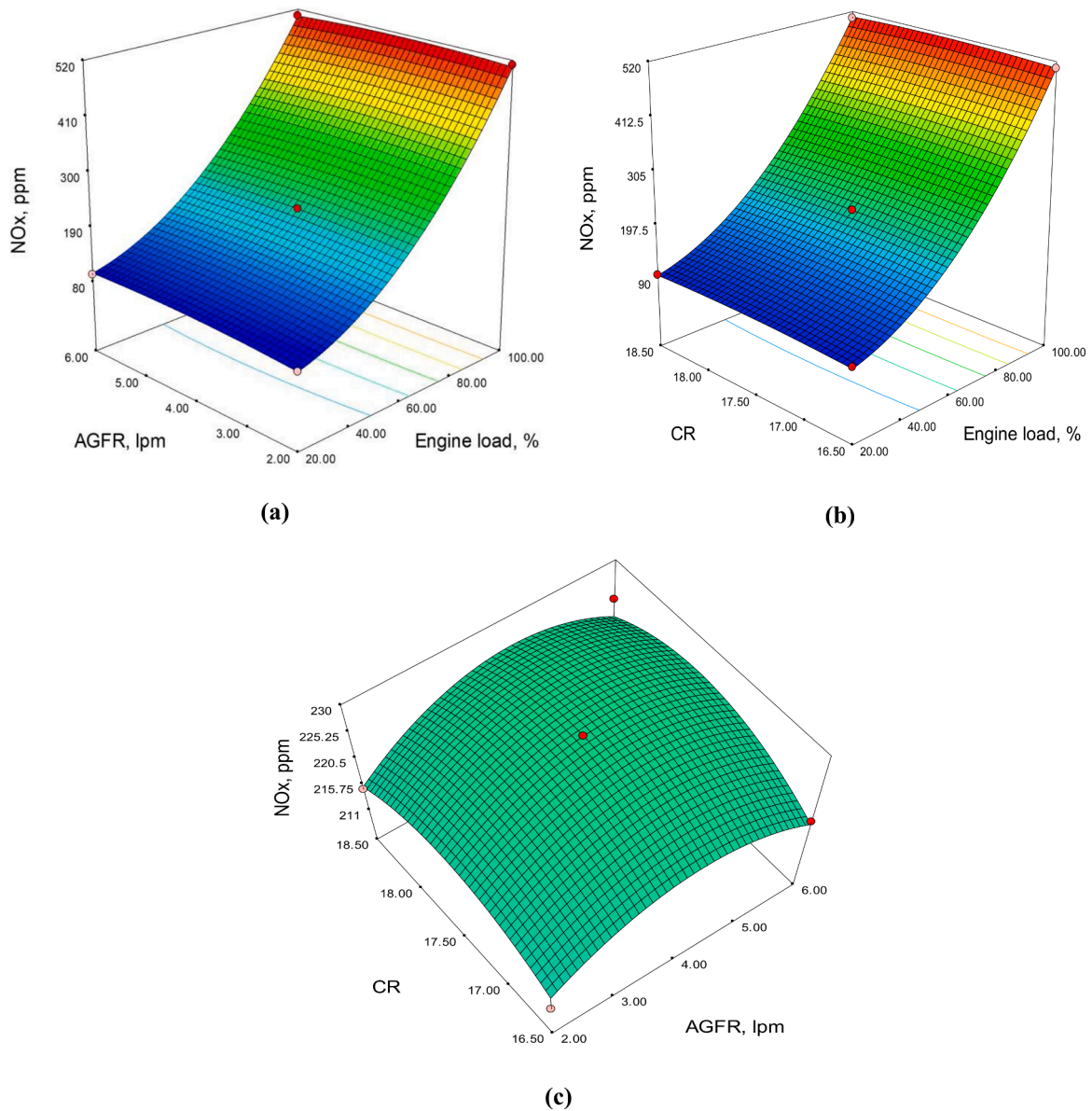


Fig. 9. Surface diagrams for NOx emission depicting the effects of; (a) CR and engine load; (b) Acetylene gas flow rate and engine load; (c) CR and Acetylene gas flow rate.

sions, resulting in environmentally friendly and productive engines in real-world applications. The desirability set in the present study is depicted in Fig. 10.

The optimized operational settings and the outcome at this optimum level is listed in Table 5. Subsequently, the engine was tested at an optimized engine operation setting. The result of engine testing at the optimized setting is also listed in Table 5.

4. Model prediction with machine learning

The data collected with experimental analysis was employed for developing the high-precision ML-based models, for effective forecasting of the engine's output. Both Tweedie and LightGBM ML techniques were used for this purpose. The procedure is depicted in Fig. 11.

It can be observed that both models performed efficiently during forecasting. In the case of the BTE model as depicted in Fig. 12a for the Tweedie-based model and Fig. 13a for LightGBM. The R^2 value for Tweedie based model was almost 1 while it was 0.86 for the based model during model training. However, when the model was tested on fresh data, the R^2 value became 0.99 and 0.8 for the Tweedie and LightGBM-based models, respectively. The prognostic errors in the models as measured with MSE were 0 and 6.09 for

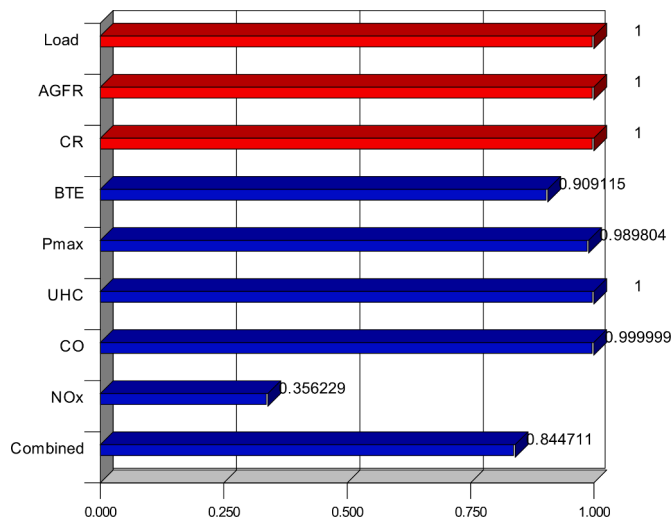


Fig. 10. Desirability levels.

Table 5
RSM optimized results, validation, and error.

Operating parameter	Optimized level	Response variable	Result at optimized setting	Experimental values	% Error
AGFR, lpm	4.48	BTE, %	27.1	26.5	2.21
Engine load, %	81.25	Pmax, bar	76.58	73	4.67
CR	18	UHC, ppm	56.2	60	6.76
		CO, ppm	104	110	5.76
		NOx, ppm	360	372	3.33

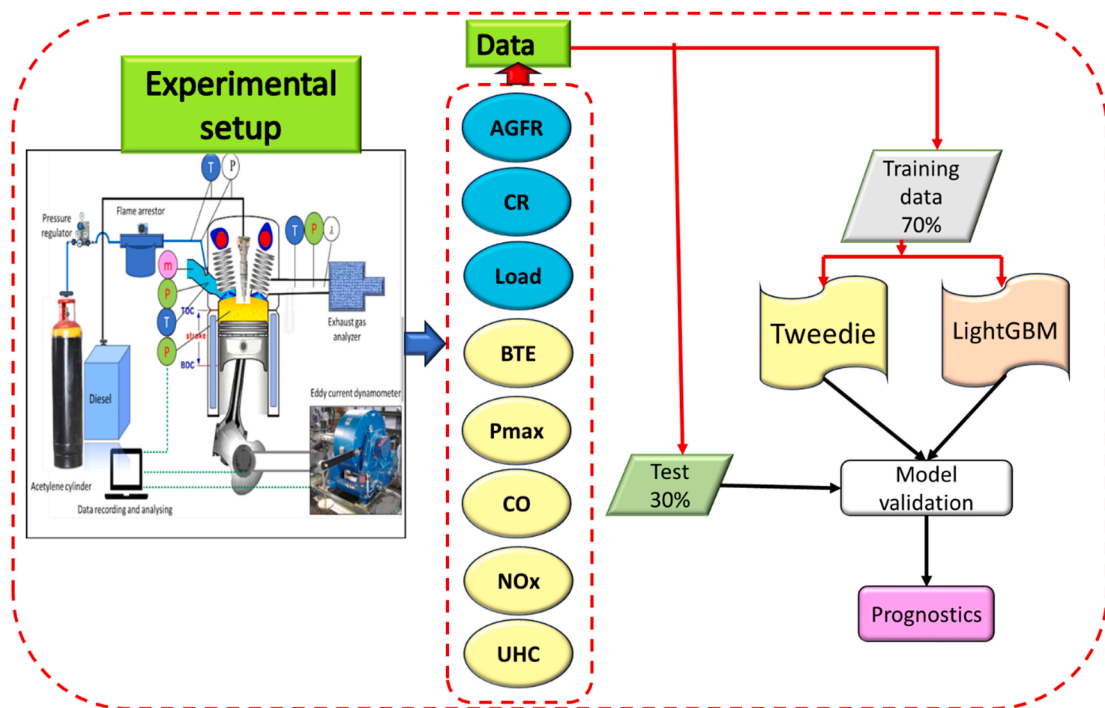


Fig. 11. Schematics of model development.

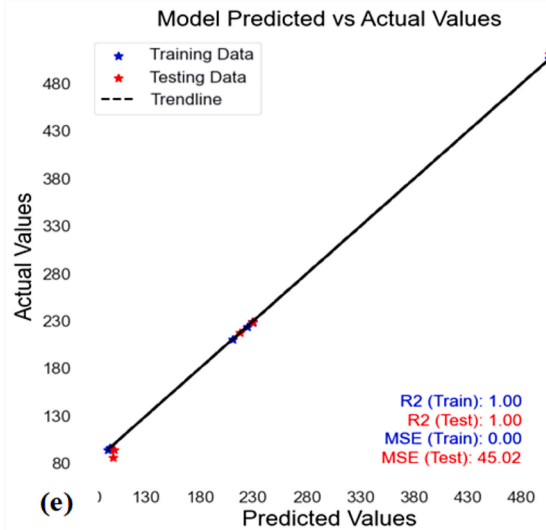
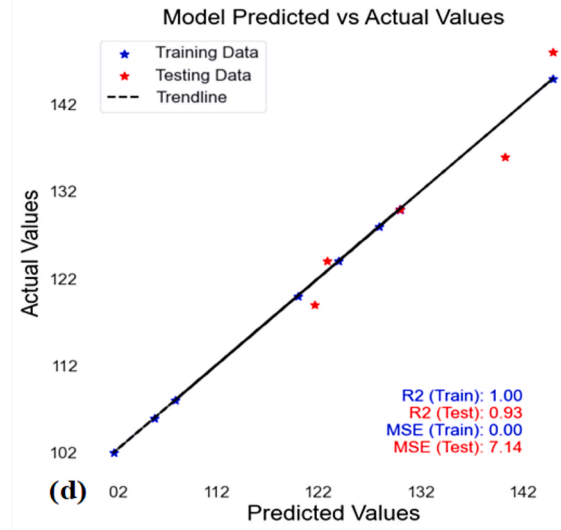
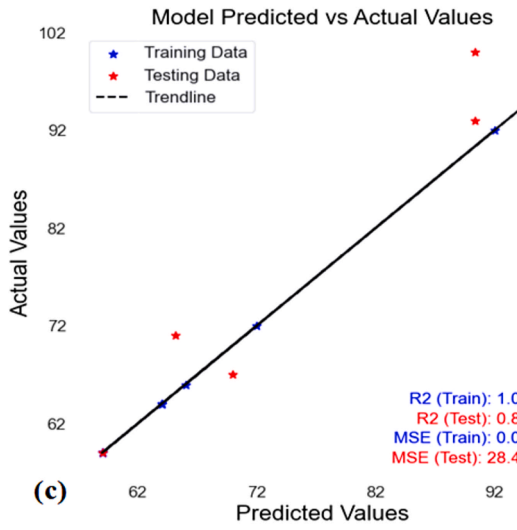
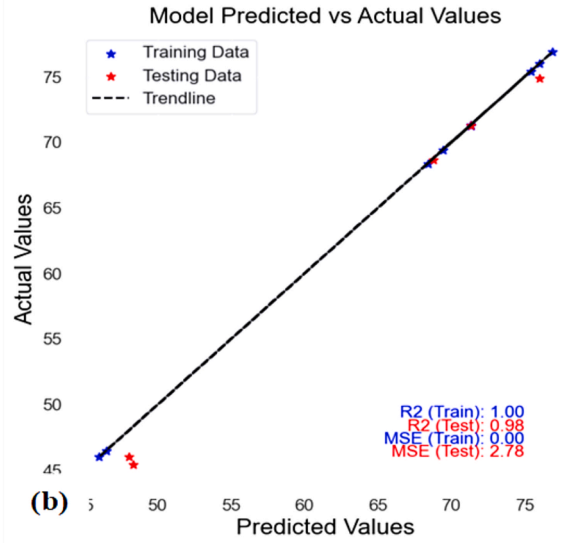
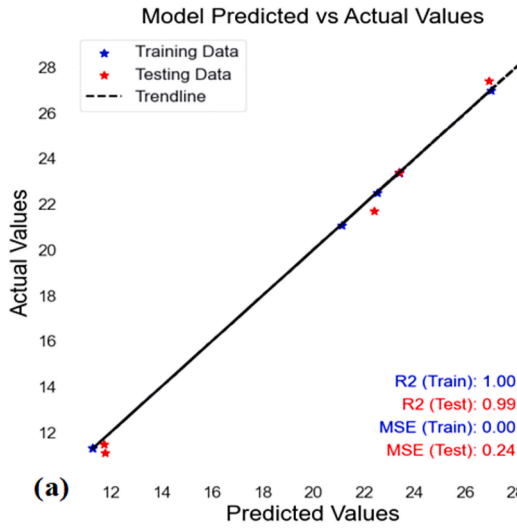


Fig. 12. Actual vs model predicted values using Tweedie ML for (a) BTE; (b) Pmax; (c) UHC emission; (d) CO emission; (e) NOx emission.

Tweedie and LightGBM-based models during the training of models. The MSE values became 0.24 and 8.5 during the model test for Tweedie and LightGBM-based models. The low model errors and high coefficient of determinates shows a robust model. However, the Tweedie model performed superior to the LightGBM model. In the case of the Pmax model, as shown in Fig. 12b for the Tweedie-based model and Fig. 13b for the LightGBM. During model training, the R^2 value for the Tweedie-based model was nearly one, whereas it was 0.74 for the LightGBM-based model. However, when the model was evaluated on new data, the R^2 values for the Tweedie and LightGBM-based models were 0.98 and 0.67, respectively. During model training, the prognostic errors in the models as determined by MSE were 0 and 28.62 for the Tweedie and LightGBM-based models, respectively. During model testing, the MSE values for Tweedie and LightGBM-based models were 2.78 and 54.37, respectively. A robust model is shown by low model errors and a high coefficient of determinates. However, the Tweedie model outperformed the LightGBM model, in this case too.

In the case of a UHC emission model, as shown in Fig. 12c for a Tweedie-based model and Fig. 13c for a LightGBM-based model. During model training, the R^2 value for the Tweedie-based model was nearly one, whereas the R^2 value for the LightGBM-based model was poor at 0.53. However, when the model is evaluated on new data, the R^2 value increases to 0.89 and 0.38 for the Tweedie and LightGBM-based models, respectively. During model training, the prognostic errors in the models, as measured by MSE, were 0 and 74.55 for the Tweedie and LightGBM-based models, respectively. During model testing, the MSE values for Tweedie and LightGBM-based models reached 28.46 and 153.89, respectively. The low model errors and high coefficient of determinates indicates a strong model. The Tweedie model, on the other hand, outperformed the LightGBM model.

As shown in Fig. 12d for a Tweedie-based model and Fig. 13d for a LightGBM-based model, for the CO emission model. During model training, the R^2 number for the Tweedie-based model was almost 1, but it was only 0.9 for the LightGBM-based model. But when the model is tested on new data, the R^2 number goes up to 0.93 for the Tweedie-based model and 0.65 for the LightGBM-based model. During model training, the Tweedie-based models had a mean squared error (MSE) of 0 and the LightGBM-based models had a mean squared error of 5.01. During model testing, the MSE values for Tweedie-based models reached 7.14 and those for LightGBM-based models hit 10.81. The high coefficient of determinates and low model mistakes show that the model is strong. The LightGBM model, on the other hand, wasn't as good as the Tweedie model. For a NOx emission model, this is shown in Fig. 12e for a Tweedie-based model and in Fig. 13e for a LightGBM-based model. During model training, the R^2 number was almost 1, for both models. It was the same during the test also. During model training, the mean squared error (MSE) for the Tweedie-based models was 0 and it was 31.14 for the LightGBM-based models. During tests, the MSE values for models based on Tweedie reached 45.02, and those for models based on LightGBM reached 52.16. The model is strong because it has a high coefficient of determinates and low model errors. On the other hand, the LightGBM model wasn't as good as the Tweedie model.

5. Conclusions

In this study, acetylene gas was employed as the primary fuel and diesel as the pilot fuel. The study effectively fine-tuned the diesel engine's operating parameters by using a desirability-based optimization method. Also, two modern machine learning approaches namely Tweedie and LightGBM were employed to develop prediction models for engine operating parameters. The main outcomes are as follows.

- o The optimized parameters estimated were 4.48 lpm for AGFR, 81.25 % for engine load, along with 18 for compression ratio. At the optimized engine setting, the model predicted BTE was 27.1 %, and the peak cylinder pressure was 73 bar. In this case, the UHC emission was 56.2 ppm, CO emission was 104 ppm, and NOx emission was 360 ppm at the optimized engine setting.
- o At the optimized engine setting, all the model-predicted results were observed well within the 2.21–6.76 % range. The R^2 value for Tweedie-based models was in the range of 0.89–1 while it was in the range of 0.38–1 in the case of the LightGBM-based models.
- o On the prediction error fronts, the mean squared error was in the range of 0.24–45.04 in the case of Tweedie-based models and it was 8.5–153.89. Even though both machine learning methods worked well, Tweedie was much better at making predictions than LightGBM-based models, both in terms of R^2 and MSE.

Finally, the optimization of operating parameters in dual-fuel diesel engines employing acetylene gas as the primary fuel offers an achievable approach to improving engine performance and lowering emissions. This study paves the way for future research into improving dual-fuel engines utilizing alternative fuels by employing intelligent approaches.

CRediT authorship contribution statement

Van Giao Nguyen: Writing – review & editing, Methodology, Conceptualization. **Brijesh Dager:** Writing – original draft, Methodology, Investigation. **Ajay Chhillar:** Writing – review & editing, Conceptualization. **Prabhakar Sharma:** Writing – original draft, Validation, Software, Methodology, Investigation, Data curation. **Sameh M. Osman:** Writing – review & editing, Methodology. **Duc Trong Nguyen Le:** Methodology, Investigation, Conceptualization. **Jerzy Kowalski:** Writing – review & editing, Methodology. **Thanh Hai Truong:** Writing – review & editing, Methodology. **Prem Shanker Yadav:** Conceptualization, Writing – review & editing. **Dao Nam Cao:** Methodology, Investigation. **Viet Dung Tran:** Methodology, Conceptualization, Writing – review & editing.

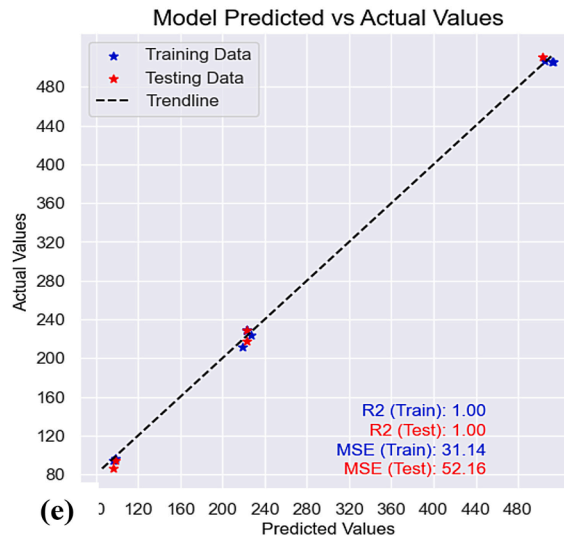
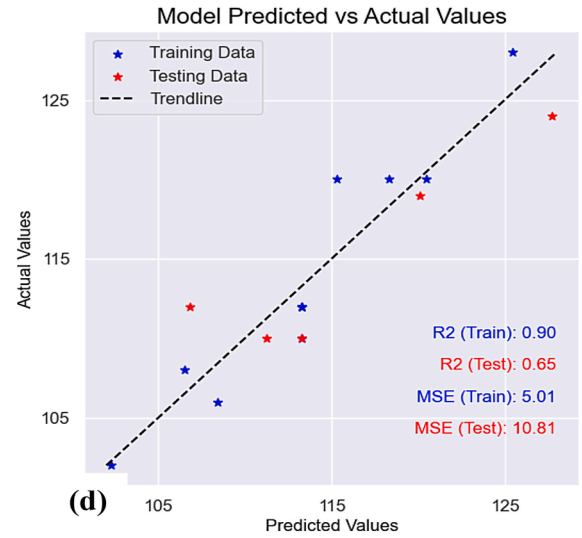
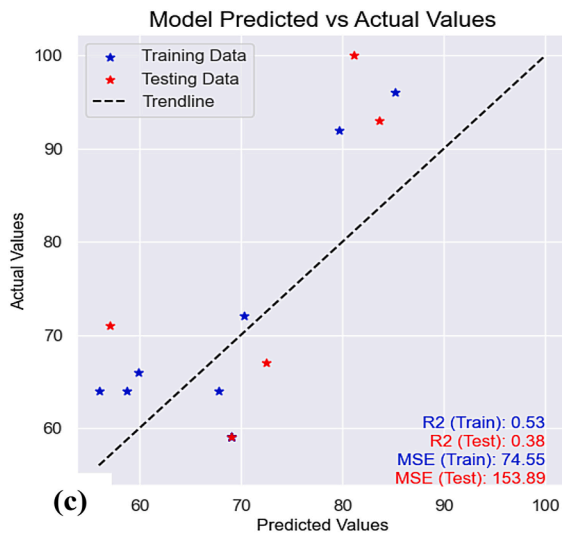
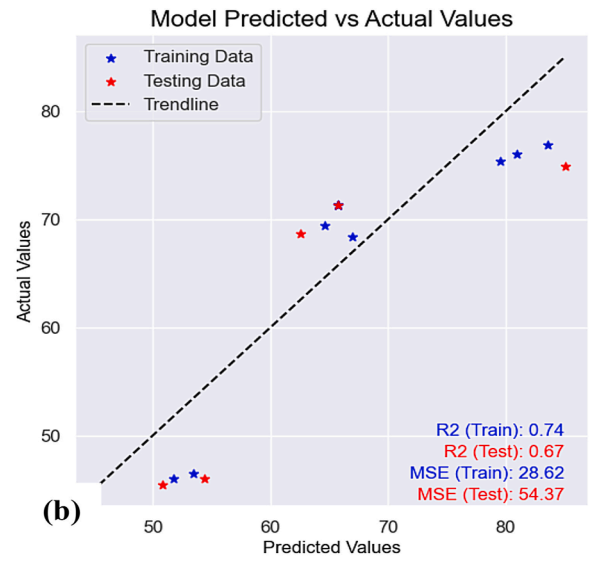
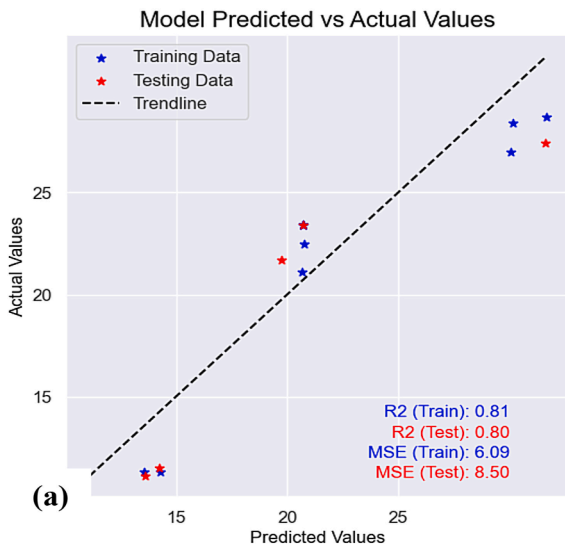




Fig. 13. Actual vs model predicted values using LightGBM for (a) BTE; (b) Pmax; (c) UHC emission; (d) CO emission; (e) NOx emission.

Declaration of competing interest

The authors declare that they have no known competing financial interests or personal relationships that could have appeared to influence the work reported in this paper.

Data availability

Data will be made available on request.

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